Amendments of the Specification:

Please replace paragraph **[0001]** with the following amended paragraph:

[0001] The present application is a divisional of U.S. Patent Application No. 10/062,609, entitled PNEUMATICALLY CONTROLLED VIBRATION DAMPER, filed January 30, 2002, by the same inventors, which claims priority from the ~~The~~ disclosure of Japanese Patent Application No. 2001-036021 filed on February 13, 2001~~including the specification, drawings and abstract is~~, which are both incorporated herein by reference in ~~its~~ their entirety.

Please replace paragraph **[0017]** with the following amended paragraph:

[0017] The abutting projection may be arranged such that the abutting projection is brought into abutting contact with the elastic connector or the abutting portion of the inner surface of the working air chamber with ~~an~~ a contact area that gradually increases with an increase in an amount of elastic deformation of the elastic connector. This arrangement makes it possible to gradually change the spring characteristic of the elastic connector by gradually increasing the restricting force applied to the elastic connector by means of the abutting projection, for example. Preferably, the abutting projection is disposed coaxially or concentrically with the elastic connector, and extends continuously or discontinuously about an elastic center axis of the elastic connector so as to be formed in a generally annular shape. This allows the elastic connector to be elastically deformed with high stability, when the abutting projection is held in its abutting state, whereby the elastic connector exhibits its desired spring characteristic with high stability.

Please replace paragraph **[0018]** with the following amended paragraph:

[0018] (5) A pneumatically controlled vibration damper according to any one of the above-indicated modes (1)-(4), wherein the mounting member and the mass member are opposed to and spaced apart from each other in a vibration input direction in which the vibration damper primary receives a vibrational load to be damped, wherein the elastic connector comprises ~~an~~ a connecting elastic body that extends outwardly from the mounting member in a direction perpendicular to the vibration input direction and is bonded at an outer peripheral portion thereof to the mass member so that the connecting elastic body primary undergoes shear deformation thereof upon a displacement of the mass member in the vibration input direction, and wherein the working air chamber is disposed between a surface of the mass member and a surface of the connecting elastic body that is opposed to the surface of the mass member in the vibration input direction. In the vibration damper according to this mode of the invention, the connecting elastic body is disposed in the vibration damper with its principle elastic axis conforming to the vibration input direction in which the vibrational load to be damped is applied, thereby increasing a spring ratio of the connecting elastic body. This arrangement enables a stable oscillation of the mass member in the vibration input direction, so that the vibration damper of this mode of the invention exhibits a desired vibration damping effect with high stability.

Please replace paragraph **[0026]** with the following amended paragraph:

[0026] The forgoing and/or other objects features and advantages of the invention will become more apparent from the following description of a preferred embodiment with reference to the accompanying drawings in which like reference numerals designate like elements and wherein:

Fig. 1 is an elevational view in vertical cross section of a pneumatically controlled vibration damper in the form of an active vibration damper constructed according to a first embodiment of the invention where the active vibration damper is in one operating state;

Fig. 2 is a graph suitable for explaining an example of a manner of static pressure control in the active vibration damper of Fig. 1;

Fig. 3 is a graph suitable for explaining another example of a manner of the static pressure control in the active vibration damper of Fig. 1;

Fig. 4 is an elevational view in vertical cross section of the active vibration damper of Fig. 1 where the active vibration damper is in another operating state;

Fig. 5 is an elevational view in vertical cross section of the active vibration damper of Fig. 1 where the active vibration damper is in yet another operating state;

Fig. 6 is a graph suitable for explaining a relationship between a static negative pressure in a working air chamber and a resonance frequency of a vibration system in the active vibration damper of Fig. 1;

Fig. 7 is a graph suitable for explaining a relationship between the static negative pressure in the working air chamber and a frequency of a maximum oscillating force in the active vibration damper of Fig. 1;

Fig. 8 is a graph suitable for explaining an example of a manner of control of active damping characteristic of the active vibration damper of Fig. 1;

Fig. 9 is a graph suitable for explaining an example of a manner of control of the oscillating force of the active vibration damper of Fig. 1;

Fig. 10 is an elevational view in vertical cross section of a pneumatically controlled vibration damper in the form of a passive vibration damper constructed according to a second embodiment of the invention, where the passive vibration damper is in one of its operating states;

Fig. 11 is an elevational view in vertical cross section of the passive vibration damper of Fig. 10 where the passive vibration damper is in the other of its operating states;

Fig. 12 is a graph suitable for explaining a relationship between a static negative pressure in ~~an~~ a working air change and a resonance frequency of a vibration system in the passive vibration damper of Fig. 10; and

Fig. 13 is a graph suitable for explaining a relationship between the static negative pressure in the working air chamber and a frequency of a maximum oscillating force in the passive vibration damper of Fig. 10, where the passive vibration damper is applied to as an active vibration damper.

Please replace paragraph **[0027]** with the following amended paragraph:

[0027] Referring first to Fig. 1, there is schematically shown an active vibration damper 10 for an automotive vehicle, which is constructed according to a first embodiment of a pneumatically controlled vibration damper of the invention, together with a drive system used for driving the active vibration damper 10. The active vibration damper 10 includes a mounting member 12 and a mass member 14 both made of metallic materials, and an elastic connector in the form of ~~an~~ a connecting elastic body 16 made of a rubber material. The connecting elastic body 16 is secured to and elastically connects the mounting member 12 and the mass member 14 so that the mass member 14 is elastically supported by the mounting member 12 via the connecting elastic body 16. The mounting member 12 and the mass member 14 are displaceable relative to each other in a vertical direction as seen in Fig. 1 in which a primary vibrational load to be damped by the present active vibration damper 10 is applied thereto.

Please replace paragraph **[0047]** with the following amended paragraph:

[0047] A graph of Fig. 2 demonstrates a change of the air pressure applied to the working air chamber 64 through the second branch conduit 80, when it is changed a ratio of a time in which the static pressure-regulating valve 70 is held in its open position, i.e., is held in communication with the atmosphere, to a period of the switching operation of the switch valve (C-VSV drive duty), under the condition where the dynamic pressure-regulating valve 72 is switched at a constant frequency. That is, a suitable adjustment of the duty ~~ration~~ ratio of the static-pressure valve 70 makes it possible to adjust the pressure of the air applied to the working air chamber 64 through the second branch conduit 80, and adjust an average air pressure value of the pressure variation induced in the working air chamber 64 by the switching operation of the dynamic pressure-regulating valve 72, i.e., a static pressure in the working air chamber 64. A graph of fig. 3 demonstrate that a suitable adjustment of a frequency of the switching operation of the static-pressure-regulating valve 70 (C-VSV drive frequency) makes it possible to adjust the static pressure applied to the working air chamber 64, likewise.

Please replace the Abstract paragraph with the following amended paragraph:

~~Disclosed is a~~ A pneumatically controlled vibration damper that includes: an elastic connector disposed between and secured to a mounting member and a mass member for elastically connecting the mass member to the mounting member, and partially defining a working air chamber closed from an external area; an air passage connected to the working air chamber for regulating an air pressure in the working air chamber from the external area; and a static pressure-regulating system operable for substantially statically regulating the air pressure in the working air chamber via the air passage, so as to induce a substantially static elastic deformation of the elastic connector for changing a spring characteristic of the elastic connector.